



Energy based modelling and control of physical systems Lecture 3

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References

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1. Stability: definitions

2. Passivity based control: Damping injection and energy shaping



Some basic notions on stability

We are considering the following class of state space model

$$\dot{x}(t) = f(x(t)),$$

 $x(0) = x_0,$
(1)

with $t \in \mathbb{R}$ and $x \in \mathbb{R}^n$. Furthermore $f : \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$. It is assumed that f(x(t)) satisfies the standard assumptions for existence and uniqueness of solutions, i.e., that f(x(t)) is Lipschitz continuous with respect to x, uniformly in t, and piecewise continuous in t.

We want to analyse the dynamics of the system

- · Do the solutions of (1) remain bounded in time?
- If yes, do they in addition converge to some equilibrium point?
- · If yes, can we say anything of the speed of convergence?
- Can we modify the solution with some external control input to impose a desired closed-loop behaviour?



We are considering the following class of state space model

$$\dot{x}(t) = f(x(t)),$$

 $x(0) = x_0,$
(2)

with $t \in \mathbb{R}$ and $x \in \mathbb{R}^n$. Furthermore $f : \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$. It is assumed that f(x(t)) satisfies the standard assumptions for existence and uniqueness of solutions, i.e., that f(x(t)) is Lipschitz continuous with respect to x, uniformly in t, and piecewise continuous in t.

Equilibrium position

An equilibrium point x^* is a solution to

 $\dot{x}(x^*) = 0$, i.e., $f(x^*) = 0$

Without loss of generality we will assume that $x^* = 0$.



An equilibrium position x = 0 of system (1) is

- 1. stable if for any $\epsilon > 0$ and $t_0 \ge 0$, there exists a $\delta(\epsilon, t_0) > 0$ such that $||x_0|| < \delta$ implies $||x(t, x_0)|| < \epsilon$ for all $t \ge t_0$,
- 2. uniformly stable if δ does not depend on t_0 ,
- 3. asymptotically stable if it is stable and for any $t_0 \ge 0$ there exists a $\Delta(t_0) > 0$ such that every solution $x(t, x_0)$ of system (1 for which $||x_0|| < \Delta$ satisfies the relation

$$\lim_{t \to \infty} \|x(t, x_0)\| \to 0 \tag{3}$$

4. uniformly asymptotically stable if it is uniformly stable, Δ does not depend on t_0 , and relation (3) holds uniformly with respect to t_0 and x_0 in the domain $t_0 \ge 0$, $||x_0|| < \Delta$,





An equilibrium position x = 0 of system (1) is

- 1. globally asymptotically stable if it is stable and relation (3) holds for any $t_0 \ge 0$ and x_0 ,
- 2. uniformly globally asymptotically stable if it is uniformly stable and relation (3) holds for any $t_0 \ge 0$ and x_0 uniformly relative to t_0 and x_0 in the domain $t_0 \ge 0$, $x_0 \in K$, where K is arbitrary compact in the x-space,
- 3. exponentially asymptotically stable if there exist positive constants Δ , M, and α such that every solution $x(t, x_0)$ of system (1), for which $||x_0|| < \Delta$, satisfies the relation

$$\|x(t, x0)\| < M \|x_0\| e^{-\alpha(t-t_0)}$$
(4)

for all $t \ge t_0 \ge 0$, and

4. globally exponentially asymptotically stable if there exist positive constants *M* and α such that relation (4) holds for $t \ge t_0 \ge 0$ and arbitrary $t_0 \ge 0$ and x_0 .



Some notions on stability





Figures taken from: http://www.math24.net/





How do we analyse stability?

Lyapunov stability theory

- * Lyapunov's direct method (second method) \rightarrow non-linear systems
- * Lyapunov's indirect method (first method) \rightarrow linear systems

Lyapunov's direct method allows to determine the stability of a system without explicitly integrating the differential equations. The method is a generalization of the idea that if there is some "measure of energy" in a system, then we can study the rate of change of the energy of the system to ascertain stability.





Let B_{ϵ} be a ball of size ϵ around the origin, $B_{\epsilon} = x \in \mathbb{R}^{n} : x < \|\epsilon\|$.

Positive definite function

A continuous function $V : \mathbb{R}^n \to \mathbb{R}$ is a locally positive definite function if V(0) = 0 and for $x \in B_{\epsilon}, x \neq 0 \to V(x) > 0$. If B_{ϵ} is the whole state space, then V(x) is globally positive definite.

A positive definite function is like an energy function.



Theorem: Lyapunov stability Let V(x) be a non-negative function with continuous partial derivatives such that

- V(x) is positive definite on B_{ϵ} , and $\dot{V} \leq 0$ locally in x and for all t, then the origin of the system is locally stable (in the sense of Lyapunov).
- If in addition $V(x) \to \infty$ when $||x|| \to \infty$, then the system is globally stable.

Theorem: Asymptotic stability Let V(x) be a non-negative function with continuous partial derivatives such that

- V(x) is positive definite on B_{ϵ} , and $\dot{V} < 0$, $\forall x \in B_{\epsilon}/\{0\}$ and V(0) = 0 locally in x and for all t, and then the origin of the system is locally asymptotically stable.
- If in addition $V(x) \to \infty$ when $||x|| \to \infty$, then the system is globally asymptotically stable.



Lyapunov's direct method





Figure : taken from: http://www.math24.net/

For physical systems: relate the physical energy with Lyapunov functions





What about the stability of passive systems?



Consider the system

$$\dot{x}(t) = f(x(t), u(t)), \quad y(t) = h(x(t), u(t)),$$
(5)

with $t \in \mathbb{R}$, $x \in \mathbb{R}^n$, $u \in \mathbb{R}^m$ and $y \in \mathbb{R}^p$. Furthermore $f : \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$ and $h : \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^p$. Let us addition define the **supply rate** w(t) = w(u(t), y(t)),

$$\int_0^t |w(u(\tau),y(\tau))d\tau| < \infty$$

Dissipative systems

The system (5) is said to be dissipative if there exists a so-called storage function $V(x) \ge 0$ such that the following dissipation inequality holds:

$$V(x(t)) \leq V(x(0)) + \int_0^t w(u(\tau), y(\tau)) d\tau$$

along all possible trajectories of (5) starting at x(0), for all x(0), $t \ge 0$.



Passive systems

Passive systems

Suppose that the system (5) is dissipative with supply rate $w(u, y) = u^T y$ and storage function V(x(t)) with V(0) = 0; i.e. for all $t \ge 0$ we have that

$$V(x(t)) \leq V(x(0)) + \int_0^t u(\tau)^\top y(\tau) d\tau,$$

Then the system is passive.

Or, equivalently

$$\dot{V}(x(t)) \leq u^{\top} y$$

And in the absence of of an external input:

 $\dot{V}(x(t)) \leq 0 \rightarrow$ Passive systems are Lyapunov stable by definition





Questions:

- · Can we make a passive system asymptotically stable?
- · Can we increase the convergence rate to the origin?
- · Can we shift the stable equilibrium?



Let us consider system arising from some physical energy model. We then usually have

$$H(t) = H(t_0) + \underbrace{\int_0^t u(\tau)y(\tau)d\tau}_{\text{supplied energy}} - \underbrace{\int_0^t S(x(\tau))d\tau}_{\text{dissipated energy}}$$

So if H(x) qualifies as a Lyapunov function and S(x) vanishes at x = 0 (and only in x = 0), then the system is asymptotically stable!

So why do we need the control then?





- What if S(x) vanishes for some $x \neq 0$ or S(x) = 0?: damping injection,
- What if we want to increase the rate of convergence?: damping injection,
- What if we want to stabilize at some different equilibrium point, $x = x^*$, $x^* \neq 0$: Energy shaping





1. Stability: definitions

2. Passivity based control: Damping injection and energy shaping





- What if S(x) vanishes for some $x \neq 0$ or S(x) = 0?: damping injection,
- What if we want to increase the rate of convergence?: damping injection,
- What if we want to stabilize at some different equilibrium point, $x = x^*$, $x^* \neq 0$: Energy shaping



Stabilization of passive systems: Damping injection

Consider the energy balance equation of a passive system:

$$H(t) = H(t_0) + \underbrace{\int_0^t u(\tau)y(\tau)d\tau}_{\text{supplied energy}} - \underbrace{\int_0^t S(x(\tau))d\tau}_{\text{dissipated energy}} .$$

And assume that H(x) qualifies as a Lyapunov function candidate. If we select the input u = -Ky, with K a positive definite constant matrix, then the energy balance equation becomes:

$$H(t) = H(t_0) \underbrace{-K \int_0^t y^2(\tau) d\tau}_{\text{controller}} - \underbrace{\int_0^t S(x(\tau)) d\tau}_{\text{dissipated energy}},$$

$$H(t) = H(t_0) - \underbrace{\int_0^t \left(Ky^2(\tau) d\tau + S(x(\tau))\right) d\tau}_{\text{dissipated energy}}.$$



Example: mass-spring-damper system



Let us consider a simple linear translational MSD system:



Constitutive relations

 $F_{s} = F_{in}$ $F_{B} = Bv_{B}$ $p = Mv_{M}$ $q = K^{-1}F_{K}$

Dynamic relations

$$F_M = rac{dp}{dt}$$
, or in integral form
 $v_K = rac{dq}{dt}$, or in integral form

$$p(t) = p(t_0) + \int_0^t F_M(\tau) d\tau$$
$$q(t) = q(t_0) + \int_0^t v_K(\tau) d\tau$$



Examples: the MSD system



$$H(x(t)) = \frac{1}{2} \frac{q}{K^{-1}}^2 + \frac{1}{2} \frac{p}{M}^2 \ge 0, \qquad H(0) = 0.$$

Hence H qualifies as a storage function and as a candidate Lyapunov function. Now,

$$H(t) = H(t_0) + \int_0^t F_{in}(\tau) \mathbf{v}_{\mathbf{M}}(\tau) d\tau - \int_0^t B \mathbf{v}_{\mathbf{M}}(\tau)^2 d\tau.$$

The system is passive if we choose $u = F_{in}$ and $y = v_M$, and furthermore, if we select u = -Ky, $(F_{in} = -Kv_M)$, then

$$H(t) = H(t_0) - \int_0^t \left(K v_M^2(\tau) + B v_M^2(\tau) \right) d\tau$$
$$= H(t_0) - \int_0^t \underbrace{\left(K + B \right)}_{B'} v_M^2(\tau) d\tau$$

We have changed (increased) the system's natural damping.



Consider the energy balance equation of a passive system:

$$\underbrace{H(t) - H(t_0)}_{\text{stored energy}} = \underbrace{\int_0^t u(\tau)y(\tau)d\tau}_{\text{supplied energy}} - \underbrace{\int_0^t S(x(\tau))d\tau}_{\text{dissipated energy}}$$

Assume that we want to change the closed-loop equilibrium to some forced (controlled) equilibrium $x = x^*$. In that case $H(x^*) \neq 0$, hence H(x) can no longer be used as Lyapunov function!

We need to consider a new Lyapunov function candidate



Let us consider the energy balance equation and assume we have no dissipation

$$H(t) - H(t_0) = \underbrace{\int_0^t u(\tau)y(\tau)d\tau}_{\text{controller}}$$

The idea is to construct a new (closed-loop) energy function, by using the (state) feedback $u = \beta(x)$

$$H_d(x,x^*) = H(x) - \int_0^t \beta(x(\tau))y(\tau)d\tau$$

such that H_d , with $H_d(x^*) = 0$ qualifies as a Lyapunov function for the system.



Stabilization of passive systems: Energy shaping

If this function exist (yes! why should it exist?) it will be a state function such that

$$H_a(x,x^*) = -\int_0^t eta(x(au)) y(au) d au$$

Hence, $H_d(x) = H(x) + H_a(x)$. The time derivative of $H_d(x)$ along the trajectories of the system is given by

$$\dot{H}_{d} = \dot{H} + \dot{H}_{a} = \dot{H} + \frac{\partial H_{a}}{\partial x}^{\top} \dot{x}$$
$$\Rightarrow \frac{\partial H_{a}}{\partial x}^{\top} \dot{x} = -\beta(x)y$$

Hence, for dynamical systems of the form $\dot{x} = f(x, u)$, y = h(x), in order to the function H_a to exist, the following PDE should be satisfied

$$\frac{\partial H_{\mathsf{a}}}{\partial x}^{\top}(f(x,\beta(x))) = -\beta(x)h(x)$$



Some remarks

• Energy shaping requires the solution of a PDE: the matching equation. Not an easy task for general non-linear systems

$$H_a(x,x^*) = -\int_0^t \beta(x(\tau))y(\tau)d\tau$$

- The existence of solutions for the PDE is strongly related with the existence of physical invariants. In the case of port-Hamiltonian systems: Casimir functions.
- For systems arising from physical applications the energy shaping technique has been proven to be a powerful stabilization method.



Example: RLC circuit

Let us consider a simple linear RLC circuit:



Dynamic relations

$$u_L = rac{d\phi}{dt},$$
 or in integral form
 $I_C = rac{dQ}{dt},$ or in integral form

Constitutive relations

$$u_{s} = V_{in}$$
$$u_{r} = RI_{r}$$
$$\phi = LI_{L}$$
$$Q = Cu_{C}$$

$$\phi(t) = \phi(t_0) + \int_0^t u_L(\tau) d\tau$$
$$Q(t) = Q(t_0) + \int_0^t l_C(\tau) d\tau$$



The state space model

$$\frac{dQ}{dt} = \frac{\phi}{L}$$
$$\frac{d\phi}{dt} = -\frac{Q}{C} - R\frac{\phi}{L} + V_{in}$$

with state variables $x = [Q, \phi]$, output $y = \frac{\phi}{L} = \frac{x_2}{L}$ and input V_{in} . The energy of the system is given by

$$H(x) = \frac{1}{2}\frac{x_1}{C}^2 + \frac{1}{2}\frac{x_2}{L}^2$$

* If $V_{in} = 0$, the natural equilibrium is x = (0, 0). If on other hand $V_{in} = V^*$, the forced equilibrium point is $x = (x_1^*, 0)$, with $x_1^* = CV^*$.



The matching equation becomes

$$\frac{\partial H_a}{\partial x}^{\top} (f(x,\beta(x))) = -\beta(x)h(x)$$
$$\frac{\partial H_a}{\partial x_1} \frac{x_2}{L} - \frac{\partial H_a}{\partial x_2} \left(\frac{x_1}{C} - R\frac{x_2}{L} - \beta(x)\right) = -\frac{x_2}{L}\beta(x)$$

Notice that the forced equilibrium corresponding to the x_2 coordinate already is a minimum of the physical energy H(x), hence we only need to shape the closed-loop energy in the x_1 coordinate. Hence

$$H_a = H_a(x_1)$$

and the matching equation becomes

$$\frac{\partial H_a}{\partial x_1} \frac{x_2}{L} = -\frac{x_2}{L} \beta(x)$$

Hence, the function H_a exists if the feedback is chosen as β





Beautiful!

The matching equation (PDE) is automatically solved for any $H_a = H_a(x_1)$ provided that the state feedback is of the form $\beta(x) = -\frac{\partial H_a}{\partial x_1}$.

• It only remains to select $H_a(x_1)$ such that $H_d = H + H_a$ has a minimum at $x^* = (x_1^*, 0)$.

Recall that the open-loop energy function is

$$H(x) = \frac{1}{2} \frac{x_1}{C}^2 + \frac{1}{2} \frac{x_2}{L}^2$$

Hence if we chose

$$H_a(x_1) = \frac{1}{2C_a} x_1^2 - \left(\frac{1}{C_a} + \frac{1}{C}\right) x_1 x_1^*$$

The closed-loop energy function $H_d = H + H_a$

$$H_d(x) = \frac{1}{2} \frac{(x_1 - x_1^*)^2}{(C + C_a)^2} + \frac{1}{2} \frac{x_2^2}{L^2}$$





 $H_d(x, x^*)$ has a minimum at $x^* = (x_1^*, 0)$ if and only if $C_a > -C$

The resulting controller is

$$u = \beta(x) = -\frac{1}{2C_a}x_1^2 - \left(\frac{1}{C_a} + \frac{1}{C}\right)x_1x_1^*$$







- We have revised some concepts from passivity based control techniques: Damping injection and Energy Shaping
- We have exploited the natural passivity of the system to design stabilizing controllers
- · Works well in many applications, but we did not see the dissipation obstacle ...

What remains for the last lesson

- · Control by interconnection
- IDA-PBC
- · Its application to PHS

